



Q1 - first order system ✓

Q2 - Second order system ✓

Q3 - Error Type number Final value ✓

Q4 - Block Diagram Reduction ✓

Q5 - Control Strategies ✓

Q6 - instruction ✗

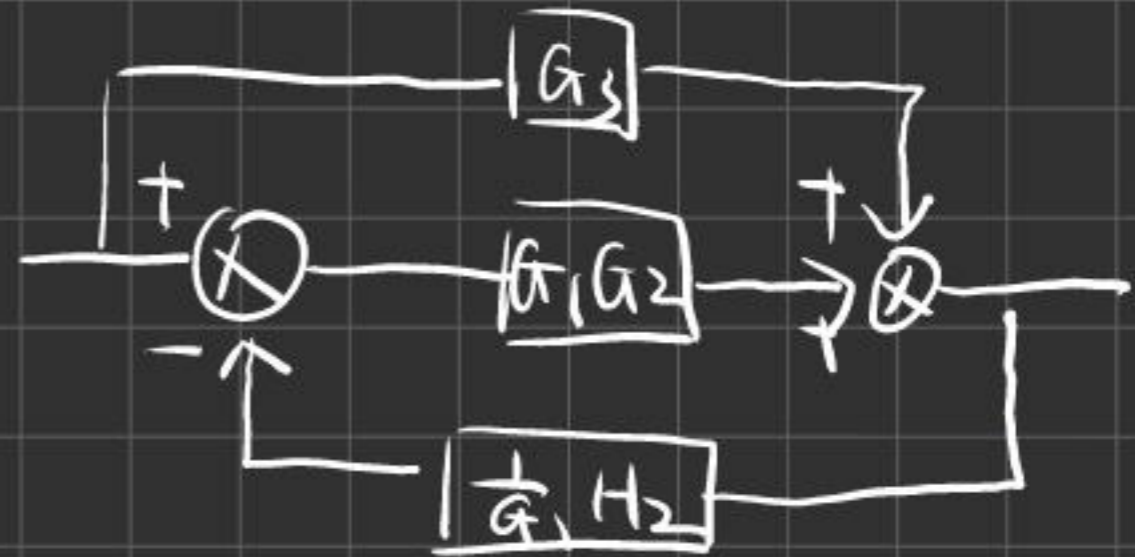
Any four of six

Block diagram reduction ✓

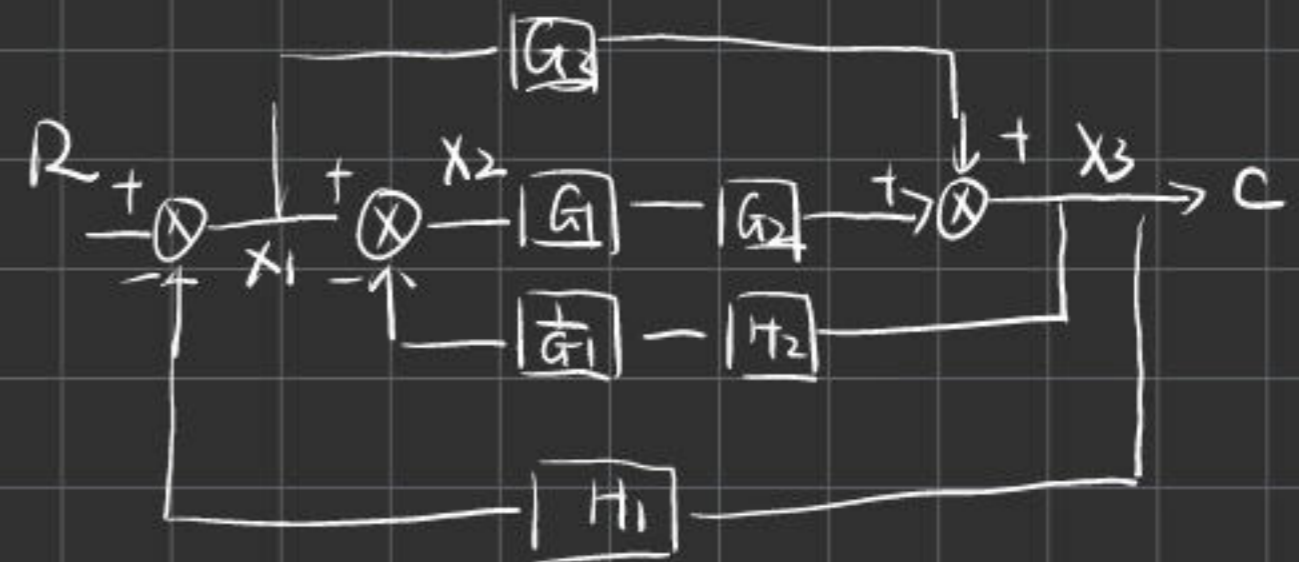
The 3 ways of reducing a block diagram.

Transformation	Original diagram	Equivalent diagram
Blocks in series $X_2 = G_1 \cdot X_1$ $X_3 = G_2 \cdot X_2 = G_2(G_1 \cdot X_1)$ $= (G_1 \cdot G_2) X_1$		
Blocks in parallel $X_2 = G_1 \cdot X_1$ $X_3 = G_2 \cdot X_1$ $X_4 = X_2 + X_3 = G_1 \cdot X_1 + G_2 \cdot X_1$ $= (G_1 + G_2) X_1$		
Eliminating a feedback loop Negative feedback $\frac{G}{1+GH}$ Positive feedback $\frac{G}{1-GH}$		

Transformation	Original diagram	Equivalent diagram
Moving a pickup point ahead of a block $X_2 = G \cdot X_1$		
Moving a pickup point behind a block $X_2 = G \cdot X_1$ $\frac{1}{G} \cdot X_2 = \frac{1}{G} \cdot G \cdot X_1 = X_1$		
Moving a summing point behind a block $X_3 = G(X_1 - X_2) = G \cdot X_1 - G \cdot X_2$		
Moving a summing point ahead of a block $X_3 = G \cdot X_1 - X_2$ $= G(X_1 - \frac{1}{G} \cdot X_2)$		
Moving Signals on Summing Blocks		
Splitting Signals in Summing Blocks		



$$\frac{G_1 G_2 + G_3}{1 + H_2 G_2}$$



$$\begin{cases} X_3 = X_1 G_3 + X_2 G_1 G_2 \\ X_2 = X_1 - X_3 \times \frac{1}{G_1} \end{cases}$$

$$\Rightarrow X_3 = X_1 G_3 + G_1 G_2 X_1 - X_3 \times H_2 \times G_2$$

$$\Rightarrow X_3 = \frac{(G_1 G_2 + G_3) X_1}{1 + H_2 G_2}$$

$$\Rightarrow \frac{G_1 G_2 + G_3}{1 + H_2 G_2} \times H_1$$

Zero order system (perfect system)

where the output is directly proportional to the input

Equation: $a_0 y = b_0 x \Rightarrow y = \frac{b_0}{a_0} x = kx$ k : static sensitivity = Gain 增益

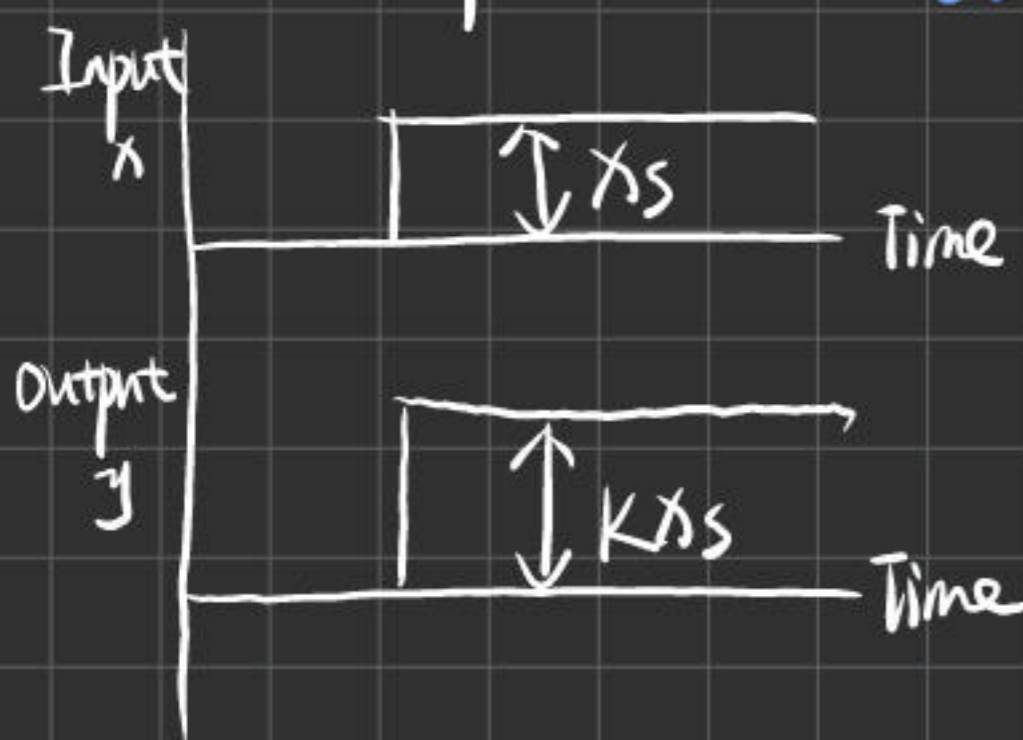
若输入变为正弦波

$x = A \sin \omega t$

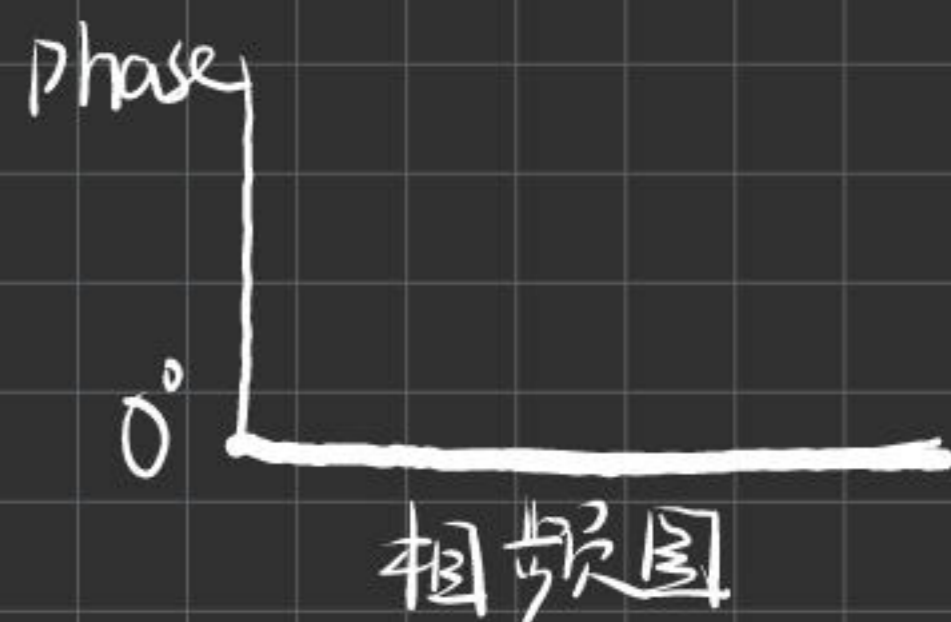
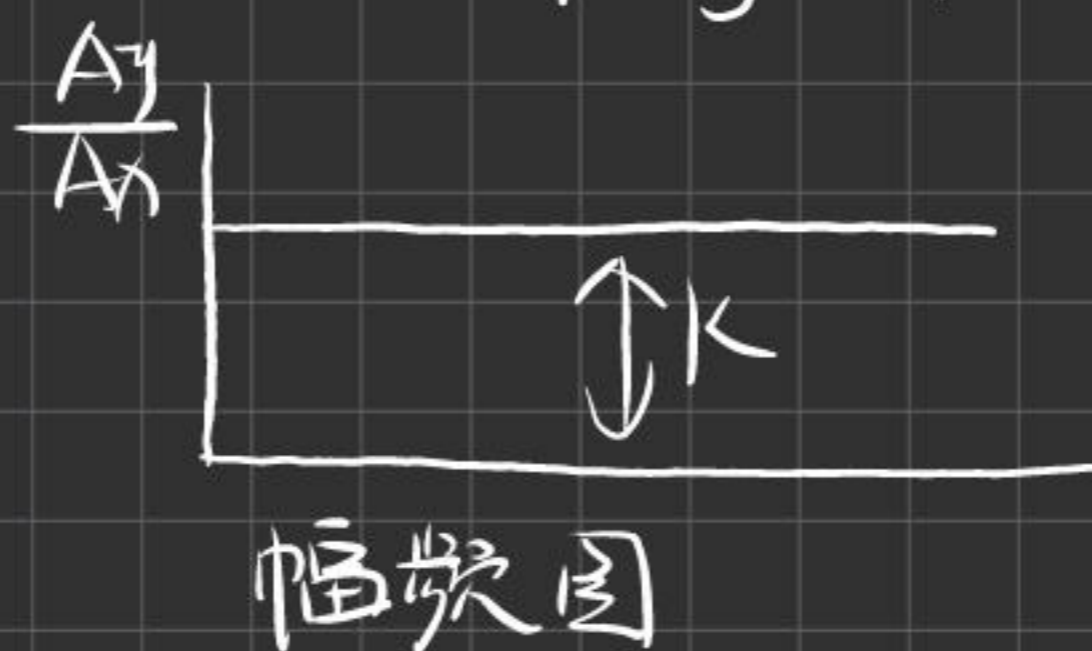
则

$y = kx = kA \sin \omega t$

Time response: 时域响应



Frequency response: 频域响应



First order system (一阶系统)

形式: $a_1 \frac{dy}{dt} + a_0 y = b_0 x \Rightarrow \frac{a_1}{a_0} \frac{dy}{dt} + y = \frac{b_0}{a_0} x$

$\Rightarrow \tau \frac{dy}{dt} + y = kx$

$\tau \times$ 输出变化率 + 输出 = $k \times$ 输入

τ : time constant 时间常数

k : Static Sensitivity (Gain)

当 output settle down,

则 rate of change of output = 0

则 输出 = $k \times$ 输入

τ 的单位是秒 (s)

k 的单位是 $\frac{y}{x}$

$\frac{y \text{ 的量纲}}{x \text{ 的量纲}}$

当 $t = \tau$ 时 $y = 0.632 k \cdot x$

When $t = \tau$, the value of output is 63.2% of the final steady value.

Laplace Transform

$$F(s) = \mathcal{L}[f(t)] = \int_0^{\infty} f(t)e^{-st} dt$$

常用 $\frac{dy}{dx} \rightarrow sY(s)$ $\frac{d^2y}{dx^2} \rightarrow s^2Y(s)$

对于 first order system ; $\tau \frac{dy}{dx} + y = kx$

拉氏变换: $\tau sY(s) + Y(s) = kX(s)$

则 transfer function: $G(s) = \frac{Y(s)}{X(s)} = \frac{k}{H\tau s}$

对于阶跃输入: $X(s) = \frac{1}{s}$ 则 $Y(s) = \frac{k}{H\tau s} \times \frac{1}{s} = \frac{k}{s} + \frac{-k\tau}{H\tau s}$

注意: $\frac{1}{s} \sim 1$, $\frac{1}{s^2} \sim t$, $\frac{1}{s + \frac{1}{\tau}} \sim e^{-\frac{t}{\tau}}$ $= k\left(\frac{1}{s} - \frac{\tau}{H\tau s}\right)$

则 $y = k \times (1 - e^{-\frac{t}{\tau}})$

$= k\left(\frac{1}{s} - \frac{1}{\frac{1}{\tau} + s}\right)$

Second order system

$$a_2 \frac{d^2y}{dt^2} + a_1 \frac{dy}{dt} + a_0 y = b_0 x$$

二阶系统标准的传递方程: $G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$ ω_n : nature frequency
 ζ : damping ratio

ω_n : nature frequency (rad/s) frequency at which the system would oscillate at if there was no damping

a measure of how well a system can respond to fast changing inputs

High $\omega_n \rightarrow$ respond quickly low $\omega_n \rightarrow$ slow respond

ζ : Damping ratio. represent friction in the system.

control the size of oscillations and how quick they decrease or die out.

$\zeta > 1$, two negative real and unequal roots
over damped response

$\zeta < 1$ under damped response

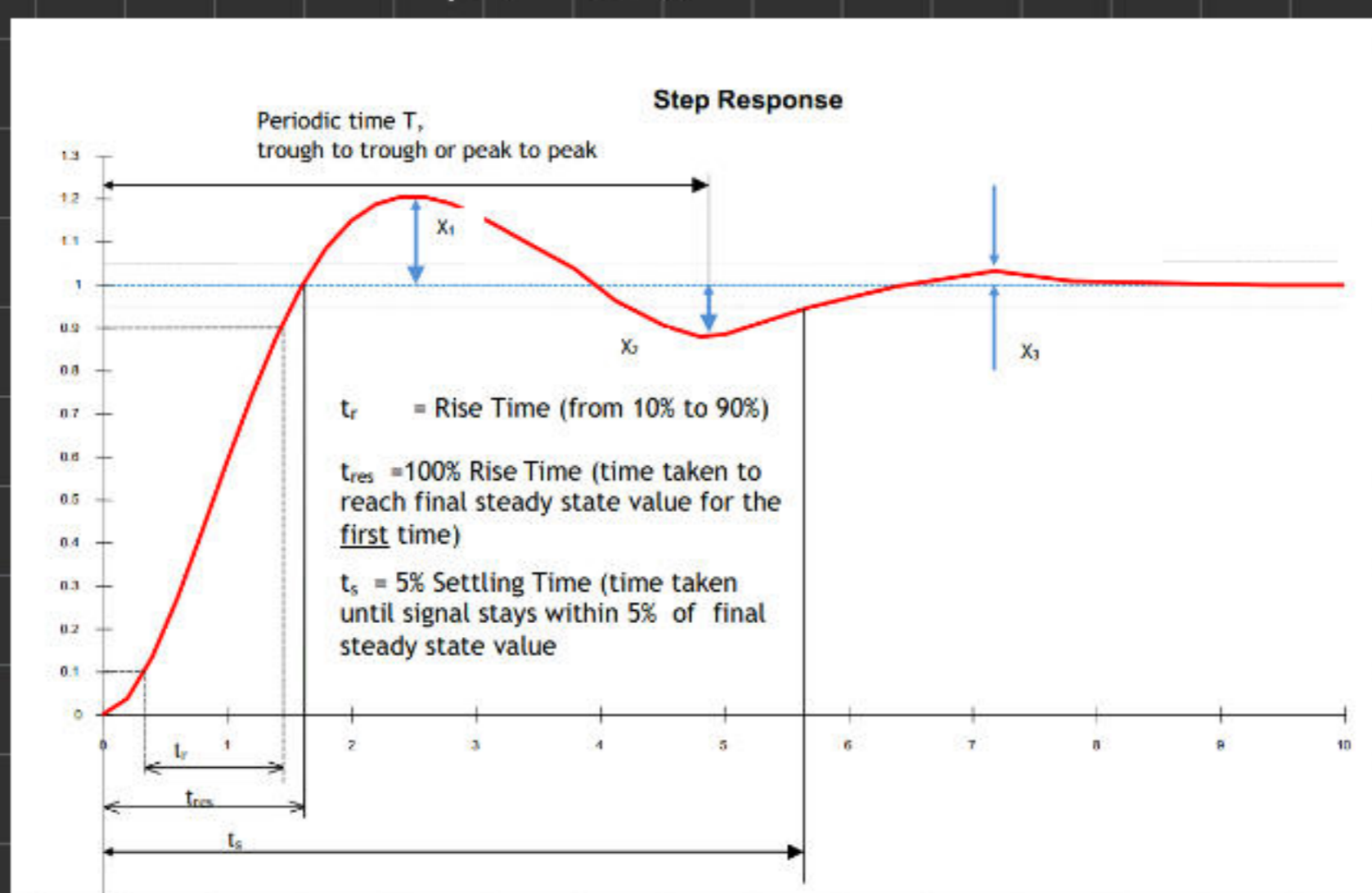
$\zeta = 1$ critically damped system

Step Response Specification

response time : (t_{res}), time taken for the system output to rise from 0% to the first over 100%

rise time : (t_r), time taken for the system output to rise from 10% to 90% of final steady value

settling time : (t_s), time taken for the system output to reach and remain within a certain percentage tolerance $\geq 2\%$, 5%



% overshoot: $e^{-\frac{\zeta \pi}{\sqrt{1-\zeta^2}}} \times 100 \%$

阻尼频率: $\omega_d = \sqrt{1-\zeta^2} \omega_n$

t_{res} (响应时间): $\frac{\pi - \varphi}{\omega_n \sqrt{1-\zeta^2}}$ $\varphi = \arccos \zeta$

2% settle time: $t_{s2\%} = \frac{4}{\zeta \omega_n}$

5% $t_{s5\%} = \frac{3}{\zeta \omega_n}$

rise time: 10% - 90% 响应时间

poles: 极点 (分母加) zeros: 零点 (分子加)

对于多项式传递函数 $G(s) = \frac{C(s)}{R(s)} = K \frac{(s-z_1)(s-z_2)\dots}{(s-p_1)(s-p_2)\dots}$

例: $G(s) = \frac{s+2}{s+5}$ 对于阶跃响应: $C(s) = \frac{1}{s} \cdot \frac{s+2}{s+5}$

$C(s) = \frac{1}{s} \frac{s+2}{s+5} = \frac{K_1}{s} + \frac{K_2}{s+5} \Rightarrow K_1 = \frac{2}{5} \quad K_2 = \frac{3}{5}$

$C(s) = \frac{2/5}{s} + \frac{3/5}{s+5}$

$c(t) = \frac{2}{5} + \frac{3}{5} e^{-5t}$
 ↓
 forced response natural response

f(t)	F(s)
Unit Impulse $\delta(t)$	1
Unit Step 1 or size A	$\frac{1}{s}$ or $\frac{A}{s}$
Unit Ramp t or At	$\frac{1}{s^2}$ or $\frac{A}{s^2}$
t^2	$\frac{2}{s^3}$
$t^n (n=1,2,3,4,5,\dots)$	$\frac{n!}{s^{n+1}}$
$\frac{df(t)}{dt}$	$sF(s) - f(0)$
$\frac{d^n f(t)}{dt^n}$	$s^n F(s) - s^{n-1} f(0) - \dots - f^{(n-1)}(0)$ Where $f^{(n)}(0) = \left[\frac{d^n f(t)}{dt^n} \right]_{t=0}$
e^{-at}	$\frac{1}{s+a}$
te^{-at}	$\frac{1}{(s+a)^2}$
$t^n e^{-at}$	$\frac{n!}{(s+a)^{n+1}}$
$\frac{1}{(b-a)} \times (e^{-at} - e^{-bt})$	$\frac{1}{(s+a)(s+b)}$
$\frac{1}{(b-a)} \times (be^{-bt} - ae^{-at})$	$\frac{s}{(s+a)(s+b)}$
$\frac{1}{ab} \left[1 + \frac{1}{(a-b)} \times (be^{-at} - ae^{-bt}) \right]$	$\frac{1}{s(s+a)(s+b)}$
$\sin \omega t$	$\frac{\omega}{s^2 + \omega^2}$
$\cos \omega t$	$\frac{s}{s^2 + \omega^2}$
$e^{-at} \sin \omega t$	$\frac{\omega}{(s+a)^2 + \omega^2}$
$e^{-at} \cos \omega t$	$\frac{s+a}{(s+a)^2 + \omega^2}$
$\frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n t \sqrt{1-\zeta^2}) \quad \zeta < 1$	$\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$
$\frac{-1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n t \sqrt{1-\zeta^2} + \phi)$ Where $\phi = \tan^{-1} \left(\frac{\sqrt{1-\zeta^2}}{\zeta} \right)$	$\frac{s}{s^2 + 2\zeta\omega_n s + \omega_n^2}$
$1 - \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n t \sqrt{1-\zeta^2} + \phi)$ Where $\phi = \cos^{-1} \zeta, \zeta < 1$	$\frac{\omega_n^2}{s(s^2 + 2\zeta\omega_n s + \omega_n^2)}$

Frequency Response

$$G(s) = \frac{Y(s)}{X(s)} \quad G(j\omega) = \left| \frac{Y}{X} \right| = \frac{K}{1 + \omega\tau j}$$

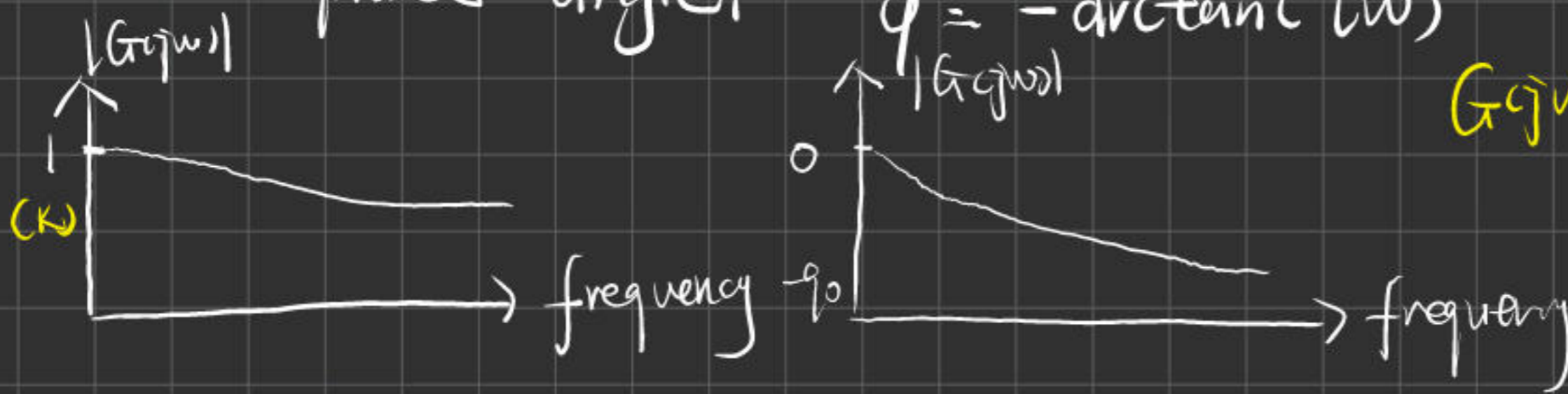
First order system:

amplitude ratio: $|G(j\omega)| = \left| \frac{Y}{X} \right| = \frac{K}{\sqrt{1 + \omega^2 \tau^2}}$ 幅频特性

phase angle:

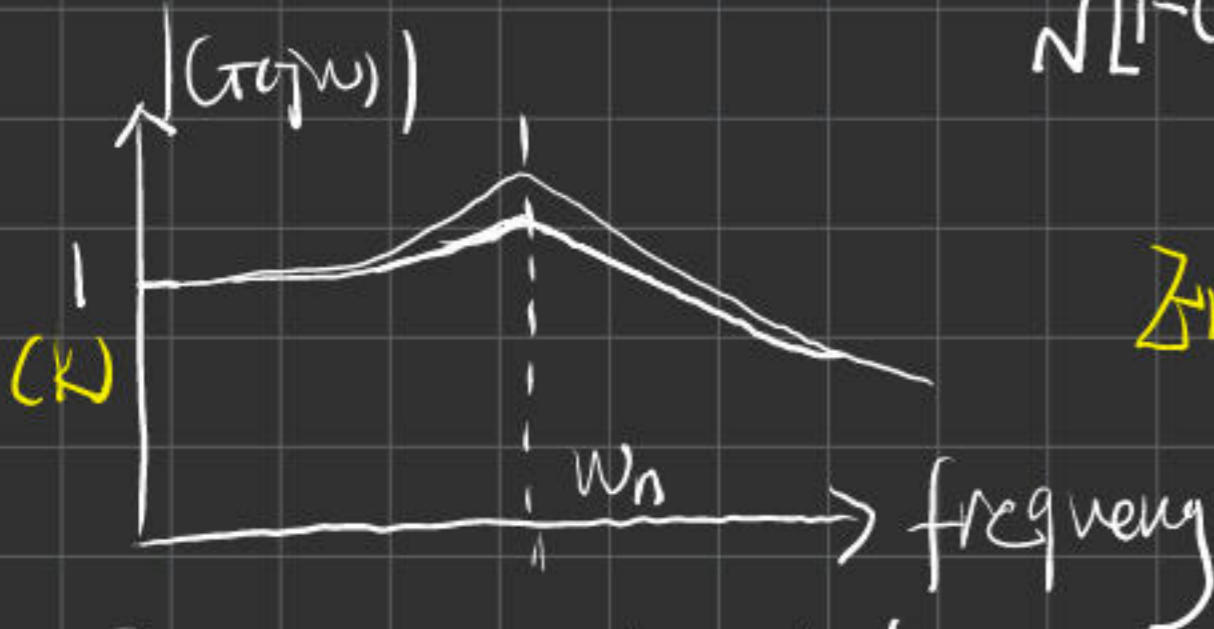
$$\phi = -\arctan(\omega\tau)$$

$$G(j\omega) = |G(j\omega)| e^{j\phi(\omega)}$$



Second order system:

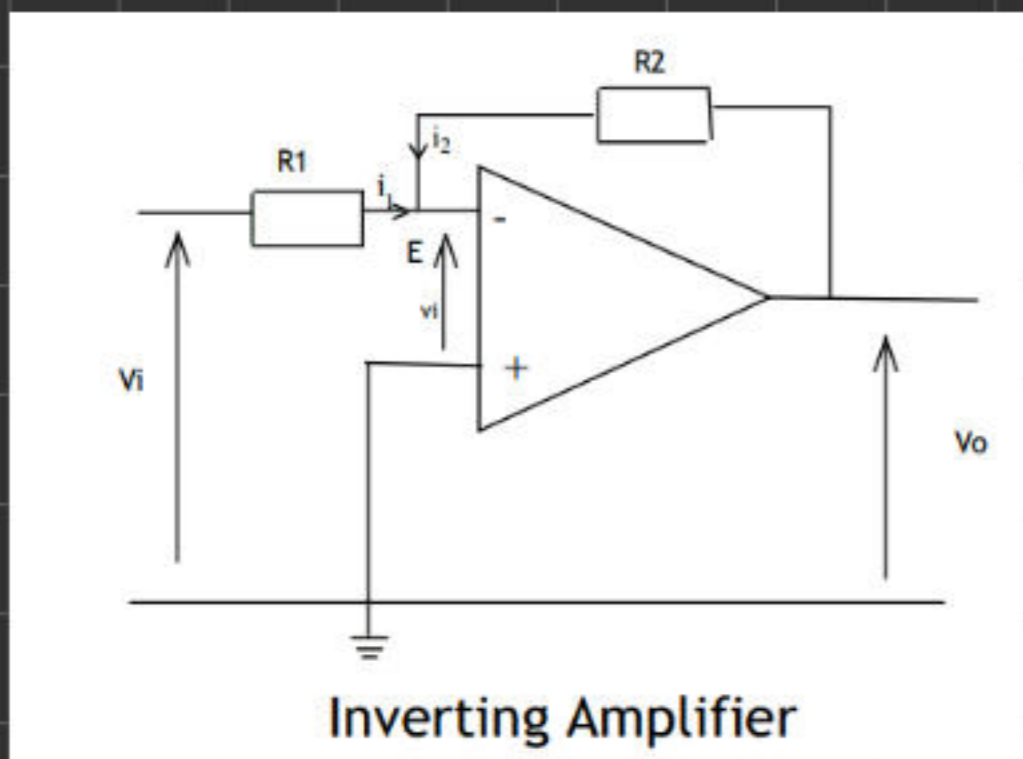
amplitude ratio: $|G(j\omega)| = \frac{K}{\sqrt{[1 - (\frac{\omega}{\omega_n})^2]^2 + (2\zeta \frac{\omega}{\omega_n})^2}}$



Error: 由频率引起的幅值变化差值

$$\text{error} = \frac{|G(j\omega)| - K}{K} = \frac{1}{\sqrt{1 - (\frac{\omega}{\omega_n})^2 + (2\zeta \frac{\omega}{\omega_n})^2}} - 1 \times 100\%$$

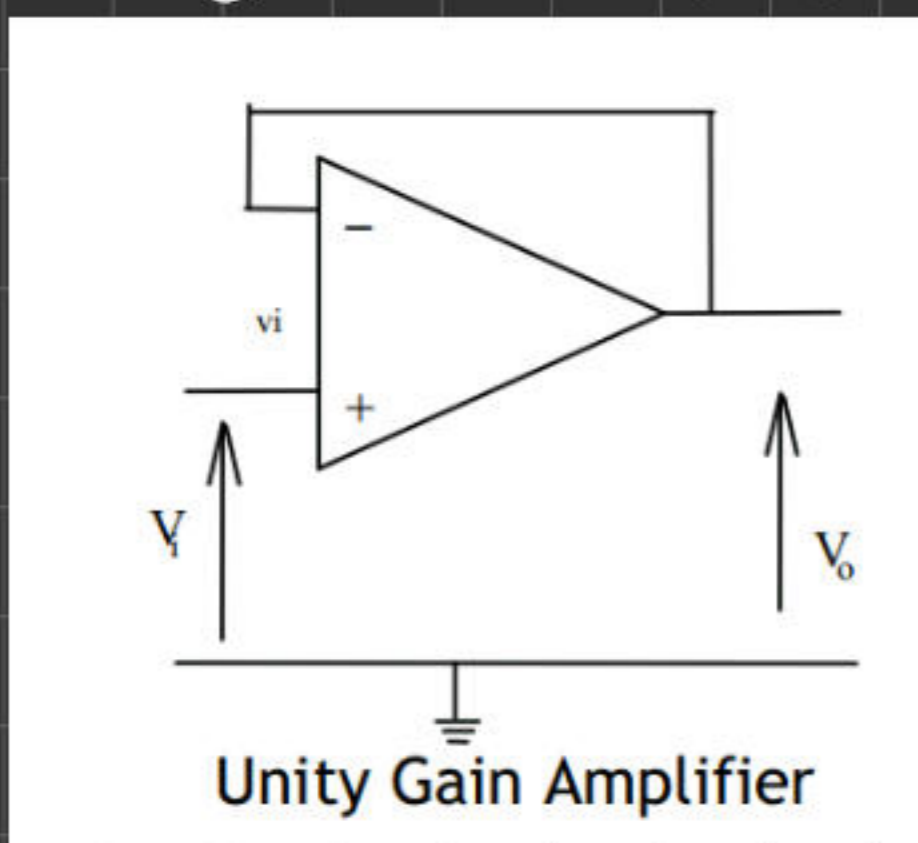
Inverting Amplifier



$$\left. \begin{aligned} i_1 &= -i_2 \\ i_1 &= \frac{V_i}{R_1} \\ i_2 &= \frac{V_o}{R_2} \end{aligned} \right\}$$

$$\frac{V_i}{R_1} = -\frac{V_o}{R_2} \Rightarrow \frac{V_o}{V_i} = -\frac{R_2}{R_1}$$

Unity Gain Amplifier

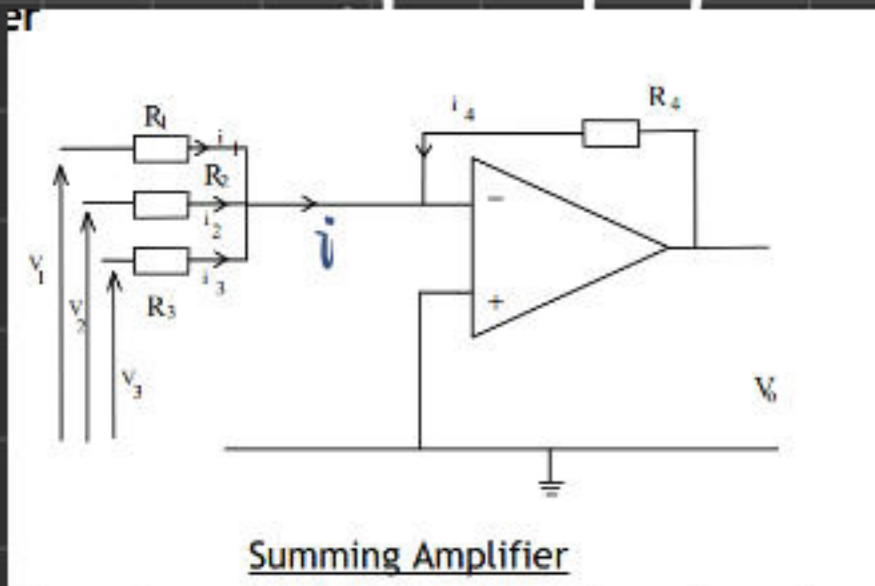


$$\left. \begin{aligned} V_o &= A V_i \\ V_i &= V_i - V_o \end{aligned} \right\}$$

$$V_o = A(V_i - V_o) \Rightarrow \frac{V_o}{V_i} = \frac{A}{1+A}$$

$$A \rightarrow \infty \Rightarrow \frac{V_o}{V_i} = 1$$

Summing Amplifier



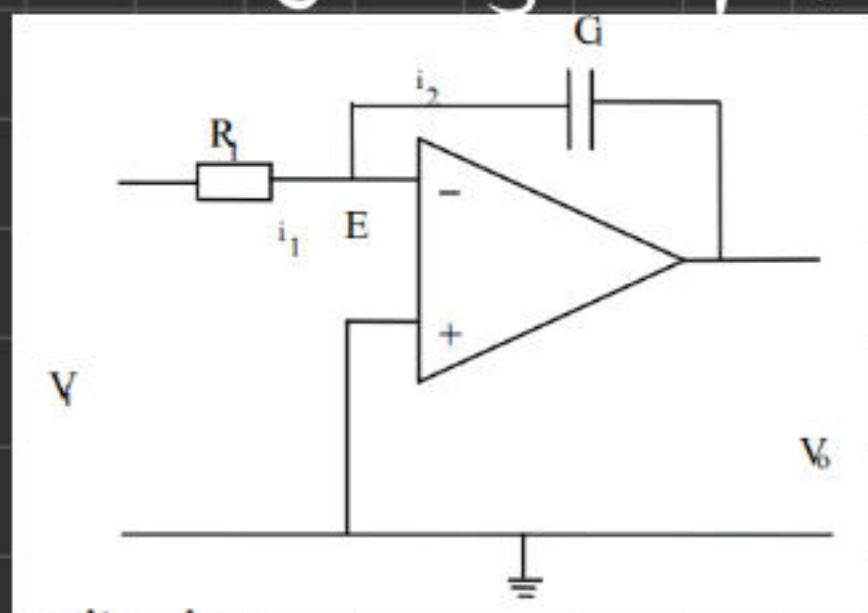
$$\begin{aligned}
 i &= -i_4 \\
 i &= i_1 + i_2 + i_3 \\
 i_1 &= \frac{V_1}{R_1} \\
 i_2 &= \frac{V_2}{R_2} \\
 i_3 &= \frac{V_3}{R_3} \\
 i_4 &= \frac{V_0}{R_4}
 \end{aligned}
 \left. \vphantom{\begin{aligned} i &= -i_4 \\ i &= i_1 + i_2 + i_3 \\ i_1 &= \frac{V_1}{R_1} \\ i_2 &= \frac{V_2}{R_2} \\ i_3 &= \frac{V_3}{R_3} \\ i_4 &= \frac{V_0}{R_4} \end{aligned}} \right\} \Rightarrow \frac{V_1}{R_1} + \frac{V_2}{R_2} + \frac{V_3}{R_3} = -\frac{V_0}{R_4}$$

$$\Rightarrow V_0 = -R_4 \left(\frac{V_1}{R_1} + \frac{V_2}{R_2} + \frac{V_3}{R_3} \right)$$

when $R_4 = R_1 = R_2 = R_3$

$$V_0 = -(V_1 + V_2 + V_3)$$

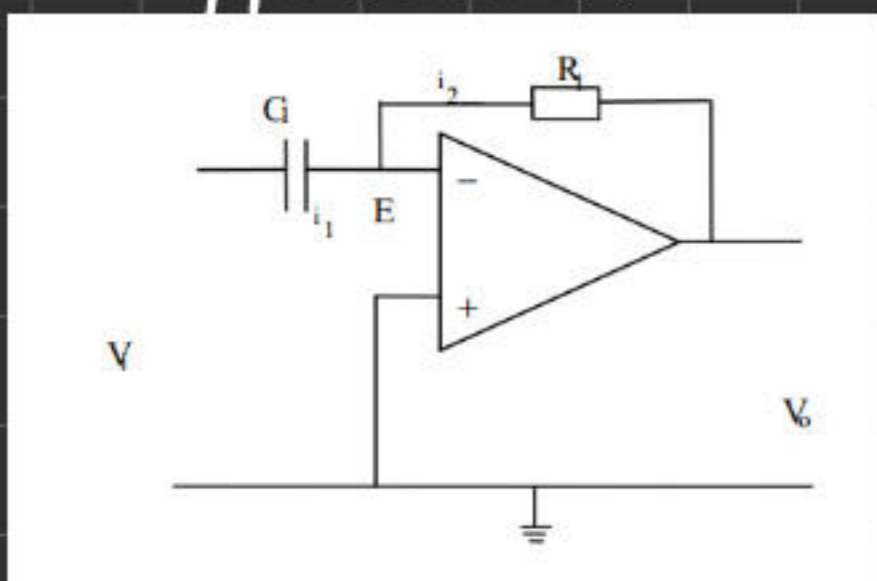
Integrating Amplifier



$$\begin{aligned}
 i_1 &= -i_2 \\
 i_1 &= \frac{V_1}{R_1} \\
 i_2 &= C \frac{dV_0}{dt}
 \end{aligned}
 \left. \vphantom{\begin{aligned} i_1 &= -i_2 \\ i_1 &= \frac{V_1}{R_1} \\ i_2 &= C \frac{dV_0}{dt} \end{aligned}} \right\} \Rightarrow \frac{V_1}{R_1} = -C \frac{dV_0}{dt} \Rightarrow \frac{dV_0}{dt} = -\frac{V_1}{R_1 C}$$

$$\Rightarrow V_0 = -\int_0^t \frac{V_1}{R_1 C} dt = -\frac{1}{R_1 C} \int_0^t V_1 dt$$

Differentiators



$$\begin{aligned}
 i_1 &= -i_2 \\
 i_1 &= C \frac{dV_1}{dt} \\
 i_2 &= \frac{V_0}{R_1}
 \end{aligned}
 \left. \vphantom{\begin{aligned} i_1 &= -i_2 \\ i_1 &= C \frac{dV_1}{dt} \\ i_2 &= \frac{V_0}{R_1} \end{aligned}} \right\} \Rightarrow C \frac{dV_1}{dt} = -\frac{V_0}{R_1} \Rightarrow V_0 = -CR_1 \frac{dV_1}{dt}$$

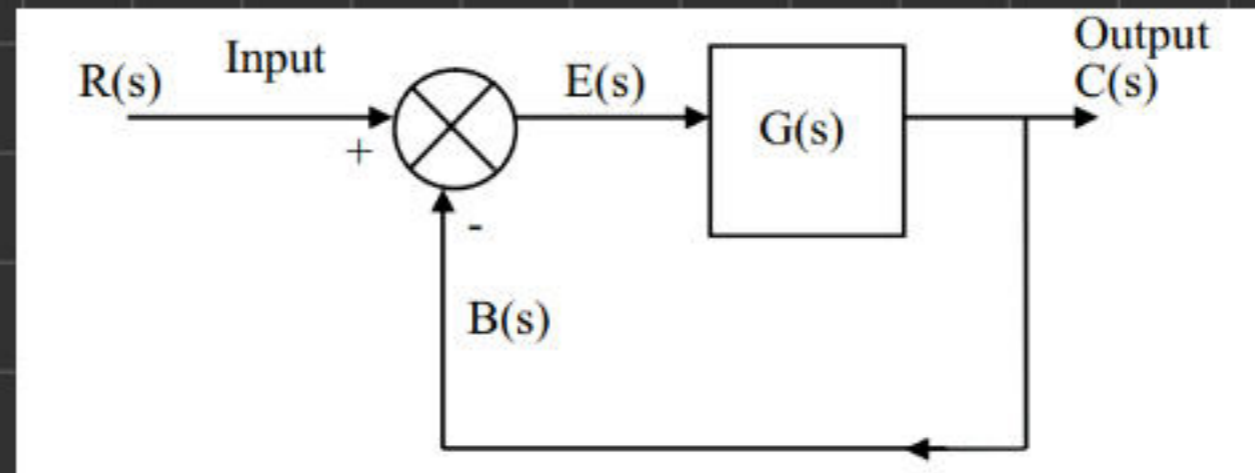
Type No and Final value and steady state error

Steady state error: $Z(s) = \frac{R(s)}{1+G(s)H(s)}$ ← 稳态误差

Final value theorem: $\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} sF(s)$

$e_{ss} = \lim_{s \rightarrow 0} sE(s) = \lim_{s \rightarrow 0} \frac{sR(s)}{1+G(s)H(s)}$

对于单位负反馈



$e_{ss} = \lim_{s \rightarrow 0} sZ(s) = \lim_{s \rightarrow 0} \frac{sR(s)}{1+G(s)H(s)}$

for step input: $R(s) = \frac{k}{s}$

$e_{ss} = \lim_{s \rightarrow 0} s \times \frac{k}{s} \times \frac{k}{1+G(s)} = \lim_{s \rightarrow 0} \frac{k}{1+G(s)} = \frac{k}{1+\lim_{s \rightarrow 0} G(s)}$

令 $K_p = \lim_{s \rightarrow 0} G(s)$

$e_{ss} = \frac{k}{1+K_p}$ K_p : positional error constant

for ramp input: $R(s) = \frac{k}{s^2}$

$e_{ss} = \lim_{s \rightarrow 0} s \times \frac{k}{s^2} \times \frac{k}{1+G(s)} = \lim_{s \rightarrow 0} \frac{k}{s+G(s)} = \frac{k}{\lim_{s \rightarrow 0} s+G(s)}$

令 $K_v = \lim_{s \rightarrow 0} sG(s)$

$e_{ss} = \frac{k}{K_v}$ K_v : velocity error constant

for acceleration input: $R(s) = \frac{k}{2s^3}$

$e_{ss} = \lim_{s \rightarrow 0} s \times \frac{k}{2s^3} \times \frac{k}{1+G(s)} = \frac{k/2}{\lim_{s \rightarrow 0} s^2 G(s)}$

令 $K_a = \lim_{s \rightarrow 0} s^2 G(s)$

$e_{ss} = \frac{k}{K_a}$ K_a : acceleration error constant

$G(s) = \frac{(s-z_1)(s-z_2)\dots}{s^\alpha (s-p_1)(s-p_2)\dots}$

for step input:	$\alpha=0$	$\alpha=1$	$\alpha=2$
	$K_p = K_p$	$K_p = \infty$	$K_p = \infty$
	$e_{ss} = \frac{k}{1+K_p}$	$e_{ss} = 0$	$e_{ss} = 0$
for ramp input:	$K_v = 0$	$K_v = K_v$	$K_v = \infty$
	$e_{ss} = \infty$	$e_{ss} = \frac{k}{K_v}$	$e_{ss} = 0$
for all input:	$K_a = 0$	$K_a = 0$	$K_a = K_a$
	$e_{ss} = \infty$	$e_{ss} = \infty$	$e_{ss} = \frac{k/2}{K_a}$

When there is a feedback block $[F]$, must use a different approach

to calculate the error: $Z(s) = [1-F(s)] \times R(s)$

$e_{ss} = \lim_{s \rightarrow 0} sZ(s) = \lim_{s \rightarrow 0} s[1-F(s)] \times R(s)$

Proportional Control: the output from the controller is proportional to the error _{input}

$$\text{proportional controller output}(t) = k_p \times e(t)$$

$$\Rightarrow M(s) = k_p Z(s)$$

PI control: To eliminate the steady state error, Integral control is added to the proportional control output.

$$\text{Integral controller output}(t) = k_i \int e(t) dt$$

$$\Rightarrow M(s) = k_i \frac{Z(s)}{s} \quad (k_i = \frac{k}{T_i})$$

PD control: the stability of a system can be improved and any tendency to overshoot can be reduced by adding derivate action.

$$\text{Derivative controller output}(t) = k_d \frac{de(t)}{dt}$$

$$\Rightarrow M(s) = k_d s Z(s) \quad (k_d = k \cdot T_d)$$

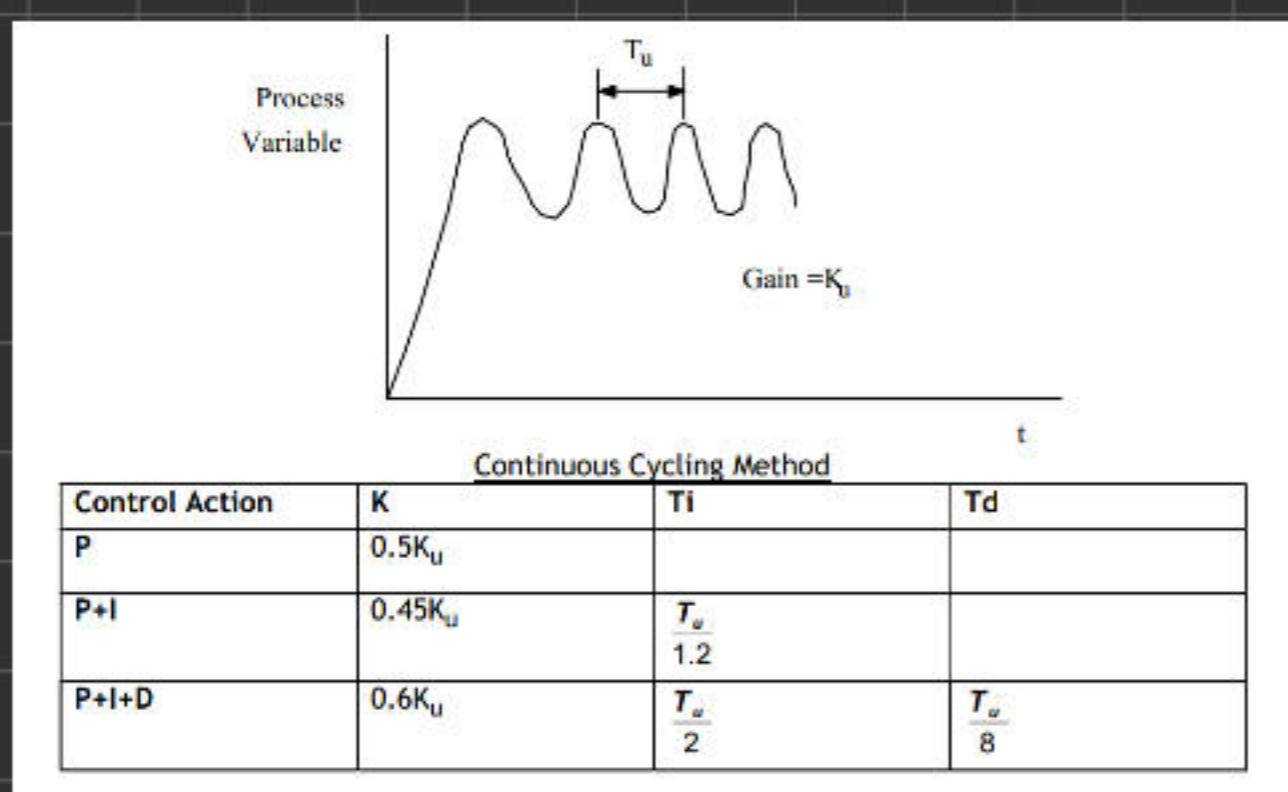
Proportional control: speed of response

Integral action: accuracy of the final steady state value

derivative action: stability

PID controller Settings

1. closed loop 'continuous cycling method'



For manual tuning process, the controller settings should be set to a minimum. The integral and derivative terms are eliminated by setting $T_d = 0$ and $T_i = \infty$.

$$\rightarrow (k_d = 0, k_i = 0)$$

Adjust the system to the desired setpoint value.

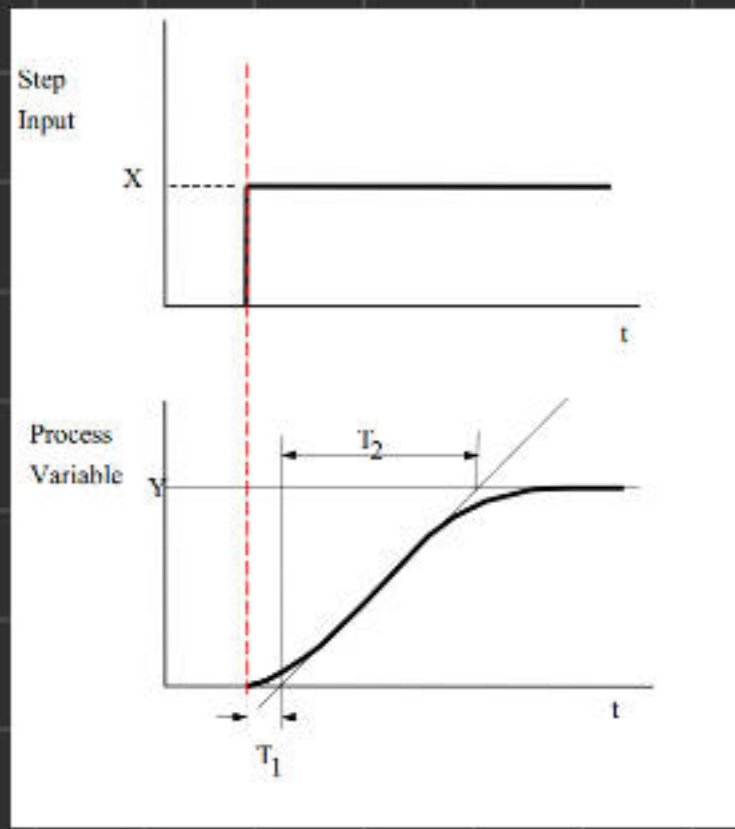
Set the proportional gain to $k_p = 1$

After the output stabilizes at the setpoint value a disturbance could be introduced to the system.

This will cause an oscillation in output.

We should obtain a sustained oscillation, so we should increase the gain until the system reached sustained oscillation.

2. Open Loop 'Reaction Curve'



Control Action	K (Proportional Gain)	Ti (Integral Time)	Td (Derivative Time)
P	$\frac{T_2}{KT_1}$		
P+I	$0.9 \frac{T_2}{KT_1}$	$\frac{T_1}{0.3}$	
P+I+D	$1.2 \frac{T_2}{KT_1}$	$2T_1$	$0.5T_1$

$$K = \frac{Y \rightarrow \Delta Y}{X \rightarrow \Delta X} \begin{array}{l} \text{change in output} \\ \text{change in input} \end{array}$$

T_1 : delay / dead time

- manual \rightarrow controller setting minimum \rightarrow setpoint value
- \rightarrow when stable, small step change \rightarrow S shaped response curve
- \rightarrow parameters are derived from the curve

